

CODE ON PERIODIC ENERGY AUDIT OF BUILDING COOLING SYSTEM

3.00 Edition

History of amendments

S/N	Brief description of changes	Revision date
1	Edition 1.0 –first issue	1 st July 2013
2	Edition 1.01 –first revision Revision to Section 6.1 Minimum OSE Standards for Energy Audit Submission New paragraph 6.1.2.1 added, to include the minimum OSE compliance	1 st April 2014
	standards for buildings that is on land within areas stipulated under the 1 st and 2 nd schedules to the BC (Environmental Sustainability) Regulations 2008	
3	Edition 2.00 - second revision Revision to Introduction - Updated Building Control (Environmental Sustainability Measures for Existing Buildings) (Amended) Regulations 2016. Revision to Section 1 Scope - Added in the list of buildings excluded: Data centres, Religious buildings, Utility buildings; and removed service apartments from the list. Revision to Section 4.1 - Paragraph 4.1.3 changed sentence: "It is only applicable for water-cooled condensers." to "It is only applicable for water-cooled chiller system." Revision to Section 5.4 - Combined paragraph 5.4.1 and 5.4.2, to include as supporting documents to be submitted via e-corenet and included Chiller(s) part load performance (at 10% interval from 100% to minimum value) from equipment supplier at operating conditions. Revision to Section 6.1 - Paragraph 6.1.3 changed from taking the largest aggregate cooling capacity of a combination of water cooled and air cooled chilled water plant to the weighted average of the air conditioning systems; and Table 6.1.4 added hospital's operating hours Revision to Section 6.2 - Footnote 1 under 6.2.7 included VSD and harmonic filter losses where applicable; and Revision to Annex A – Included harmonic filter losses (where applicable) for Plant A/B/C under Note.	1st July 2016

	The key	y changes to Annex B include:	
	a)	Added Building Address and Image; and PE/Energy Auditor signature on front cover	
	b)	New Section on Executive Summary & Recommendation	
	c)	Under building information, included paragraph for number of guest rooms for hotel/service apartment	
	d)		
	e)	New Paragraph on Description of Plant Control Strategy	
	f)	Updated table of instrumentations to include brand and calibration laboratory	
	g)	Chiller Plant Performance Analysis to show 1 week data and added various figures	
	h)		
	i)	New Section on Schedule of space operating conditions	
	j)	New Appendix on Checklist of Plant Operating Condition (for best practices)	
4	Edition	3.00 - third revision	1 st March 2020
	ensure through require pursual perform overall	n to Section 1 - added "The aim of the Periodic Energy Audit is to that the chilled water plant continues to operate efficiently hout its life cycle. The Energy Audit Code sets out the ements for periodic energy audit of a building cooling system into section 22FF of the Building Control Act. It specifies the mance standards and provides the methodology to determine the performance and heat balance of the chilled-water plant based on operating conditions."	
	Revisio		
	Revisio and Sta		
		part c-SS553: Code of Practice for Air-conditioning and Mechanical Ventilation in Buildings,	

- part d-SS554: Code of Practice for Indoor Air Quality for Air-Conditioned Buildings, and
- part e-SS591: Code of Practice for Long Term Measurement of Central Chilled Water System Energy Efficiency

Revision to paragraph 4.1.2 - include "The PE(Mech)/Energy Auditor shall check the accuracy of permanent temperature sensors with a portable calibrated temperature sensor before any data extraction for analysis and computation of performance data. The average deviation between two sensors, after taking into consideration the effects of heat gain or loss in the pipe, shall not exceed 0.07 °C as shown in example in Appendix B of OSE report."

New Section 4.2 Indoor Space Operating Conditions to include:

New paragraph 4.2.1 Energy efficiency requirements of the chiller plant shall be met without adjusting the normal operation of the building. Space conditions should be maintained within the recommended limits of SS553 and SS554 where applicable.

New paragraph 4.2.2 The OSE report shall include a 10-point spot measurements of indoor air conditions - dry-bulb temperature, relative humidity and carbon dioxide concentration and tabulated in "Schedule of Operating Conditions" of OSE template.

Revision to Section 6.1.1: the minimum OSE is 0.80kW/RT for building cooling load <500RT and included a footnote "* For project with Design Score approved before 1 Apr 2019, the minimum OSE is 0.85kW/RT

Key changes to Annex B – Energy Audit Report for Building Cooling System

- a) Amended Executive Summary & Recommendation
- b) Include minimum OSE standard to comply
- c) Amended Table 1, 2, 3, 6 & 7
- d) Amended Section 7.0 Schedule of Space Operating Conditions and included a footnote "*Any observation on over-cooling/ under-cooling and ventilating of space conditions should first be investigated and corrected before the energy audit is carry out. Refer to recommended limits of SS553 and SS 554."

- e) Added Table 8, 9 and Figure 20, 21
- f) Amended Appendix A: Checklist of Plant Operating Condition (for best practices)
- g) Added new Appendix B: Temperature Sensors Verification Plot (worked example)
- h) Added new Appendix C: Total Cooling System Efficiency (worked example)

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INTRODUCTION

The requirements governing the Periodic Energy Audit of Building Cooling Systems are stipulated in Part IIIB of the Building Control Act and the Building Control (Environmental Sustainability Measures for Existing Buildings) (Amended) Regulations 2016.

The intent of the Code on Periodic Energy Audit of Building Cooling Systems (referred to as "Energy Audit Code") is to ensure that building cooling systems are audited to meet minimum system efficiency and continue to be operated efficiently throughout its life cycles.

The Energy Audit Code is not intended to abridge safety, health, environmental or related requirements contained in other applicable laws, codes or policies administered by relevant authorities. Where there is a conflict between the requirements of the Energy Audit Code and other requirements affecting the design, construction or operation of the Chilled Water Plant in the building cooling systems, precedence shall be determined by the relevant authorities, where required.

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1. SCOPE

The aim of the Periodic Energy Audit is to ensure that the chilled water plant continues to operate efficiently throughout its life cycle.

The Energy Audit Code sets out the requirements for periodic energy audit of a building cooling system pursuant to section 22FF of the Building Control Act. It specifies the performance standards and provides the methodology to determine the overall performance and heat balance of the chilled-water plant based on actual operating conditions. The Code also specifies the qualification and experience requirements for registration and renewal as an Energy Auditor with BCA under the Act.

The provisions of the Energy Audit Code shall apply to:-

- (a) Existing buildings that have undergone Major Energy Use Change; and
- (b) New buildings that have applied for planning permission on or after 1 December 2010.

The following building types shall not be covered under the Energy Audit Code:

- a) Data centres;
- b) Religious buildings;
- c) Residential buildings (other than service apartments);
- d) Utility buildings;
- e) Industrial buildings;
- f) Railway premises;
- g) Port services and facilities; and
- h) Airport services and facilities.

The reference codes, standards and other documents referred in the Energy Audit Code shall be considered part of the requirements of the Energy Audit Code to the extent as prescribed.

2. **DEFINITIONS**

For the purpose of the Energy Audit Code, the following definitions shall apply:

BC Act Building Control Act (Chapter 29)

BCA Building and Construction Authority

Chilled Water Plant centralised air conditioning system which makes use of

chilled water as the medium for removing the heat from the building. This includes the chillers and its ancillary equipment, including pumps and cooling towers where

applicable.

Energy Audit An audit on the Operating System Efficiency of a Chilled

Water Plant.

Energy Audit Report The report of the Energy Audit in the format prescribed in

Annex B to the Energy Audit Code.

Energy Auditor A competent person registered with BCA to conduct the

Energy Audit.

Minimum Green Mark Score The minimum standard of environmental sustainability

applicable to that building, expressed in terms of a Green

Mark score.

Major Energy Use Change The installation, substantial alteration or replacement of

a/the Water-Cooled/Air-cooled Chiller(s).

Notice The notice served on the building owner by the

Commissioner of Building Control requiring an Energy Audit

to be carried out.

Operating System Efficiency (OSE) The measured system efficiency of the building's Chilled

Water Plant during its normal operating hours.

PE(Mech) a person who is registered under the Professional Engineers

Act (Cap. 253) in the mechanical engineering discipline and who has in force a practising certificate issued under that Act authorising him to engage in mechanical engineering

work.

Regulations The Building Control (Environmental Sustainability Measures

for Existing Buildings) Regulations 2013 and Building Control

(Environmental Sustainability) Regulations 2008

In instances where terms are not expressly stated in the Energy Audit Code and are defined in other referenced documents, such terms shall have the meanings as determined in those documents.

3. STATUTORY REQUIREMENTS

3.1 Act and Regulations

The following Act and Regulations have relevance:

- a. BC Act; and
- b. Regulations.

3.2 Referenced Codes and Standards

The following codes and standards have relevance:

- a. Code for Environmental Sustainability for Buildings
- b. Code on Environmental Sustainability Measures for Existing Buildings
- c. SS553: Code of Practice for Air-conditioning and Mechanical Ventilation in Buildings
- d. SS554: Code of Practice for Indoor Air Quality for Air-Conditioned Buildings.
- e. SS591: Code of Practice for Long Term Measurement of Central Chilled Water System Energy Efficiency

3.3 Responsibility

Upon receiving the Notice, the building owner shall engage a PE(Mech) or an Energy Auditor registered with BCA, to ensure that the periodic Energy Audit is carried out in accordance with the BC Act, the Regulations and the requirements set out in the Energy Audit Code. The building owner shall ensure that the OSE meets the minimum energy efficiency standard (referred to as "Minimum OSE") stipulated in section 6 of the Energy Audit Code and he/she shall undertake further measures within the time frame specified in the Notice to meet the Minimum OSE if the Energy Audit reveals otherwise.

The building owner shall submit the Energy Audit Report prepared by the PE(Mech)/ Energy Auditor in the format found in Annex B.

4. COMPLIANCE METHOD

4.1 Operating System Efficiency

- **4.1.1** The OSE of the Chilled Water Plant shall be determined by the energy audit methodology specified in section 6 of the Energy Audit Code.
- 4.1.2 The data required for the determination of the OSE shall be from the permanent instrumentations installed in compliance with the Code for Environmental Sustainability of Buildings or the Code on Environmental Sustainability Measures for Existing Buildings that is prevailing at the time of installation. The PE(Mech)/Energy Auditor shall check the accuracy of permanent temperature sensors with a portable calibrated temperature sensor before any data extraction for analysis and computation of performance data. The average deviation between two sensors, after taking into consideration the effects of heat gain or loss in the pipe, shall not exceed 0.07 °C as shown in example in Appendix B of OSE report.
- **4.1.3** The heat balance substantiating test is used to determine the quality of the measurements i.e. the uncertainty, installation and position of the instruments is within the prescribed tolerance and requirements of the instrumentations as specified in the Energy Audit Code. It is only applicable for water-cooled chiller system.

4.2 Indoor Space Operating Conditions

- **4.2.1** Energy efficiency requirements of the chiller plant shall be met under normal operating conditions of the building. Space conditions should be maintained within the recommended limits of SS553 and SS554 where applicable.
- 4.2.2 The OSE report shall include a 10-point spot measurements of indoor air conditions dry-bulb temperature, relative humidity and carbon dioxide concentration and tabulated in "Schedule of Operating Conditions" of OSE template.

5. SUBMISSION PROCEDURES

5.1. General

Building owners will receive a Notice from BCA notifying them that the submission of the Energy Audit is required. Upon receiving the Notice, the building owner shall appoint PE(Mech)/Energy Auditor to submit the Energy Audit Report within the timeframe specified in the Notice. BCA will issue Notice of Approval if the submission of Energy Audit Report is in order.

5.2 For buildings complying with the Code on Environmental Sustainability Measures for Existing Buildings

- **5.2.1** BCA may serve the 1st Notice to the building owner to submit the Energy Audit Report not less than 36 months from the date of the Notice of Approval of the as-built score.
- 5.2.2 PE(Mech)/Energy Auditor shall complete and submit the Form BCA-EB-OSE01 (Application for Approval of Operating System Efficiency of Building Cooling Systems) with supporting documents electronically via CORENET esubmission system.
- 5.2.3 The building owner may be served each subsequent Notice not less than 36 months from the date of the last Notice, for subsequent Energy Audit submissions.

5.3 For buildings complying with the Code for Environmental Sustainability for Buildings (from 2nd Edition)

- 5.3.1 BCA may serve the 1st Notice to the building owner to submit the Energy Audit Report any time after issuance of Temporary Occupation Permit (referred to as "TOP") or Certificate of Statutory Completion (referred to as "CSC"), if no TOP is issued. PE(Mech)/Energy Auditor shall complete and submit the Form BCA-EB-OSE01 with supporting documents electronically via CORENET e-submission system.
- **5.3.2** The building owner may be served each subsequent Notice not less than 36 months from the date of the last Notice, for subsequent Energy Audit submissions.

5.4 Documentary Evidence

5.4.1 The building owner and the PE(Mech)/Energy Auditor shall submit the following supporting documents electronically via CORENET e-submission system:-

- Scanned copy of BCA's notice for the submission of the Energy Audit in pdf format
- As-built drawings of chiller plant room plan layout indicating details of instruments locations
- As-built schematic drawings of the chiller plant
- 1 week raw data of the following data points in excel format (xls.) with date and time stamp: chilled water supply temperature (°C); chilled water return temperature (°C); condenser water supply temperature (°C); condenser water return temperature (°C); chilled water flow rate (I/s); condenser water flow rate (I/s); electrical power of chiller(s), chilled water pump(s), condenser water pump(s) & cooling tower(s) (kW). The excel file should include all the chart plots specified in Annex B.
- Energy Audit Report in accordance to ANNEX B
- Instruments' calibration certificates from the accredited laboratories and their factory calibration certificates from manufacturers.
- Drawings showing the details of instruments installation
- All input parameters for the permanent instrumentations (e.g. flow meter settings for pipe material, diameter, circumference, thickness, roughness, type of lining etc.)
- Calculation of the overall uncertainty of measurement of the resultant chiller plant in kW/RT to be within ±5% of the true value based on instrumentation specifications and calibration certificates.
 - 1) for central chilled water system; and
 - for individual chillers (if instrumentation are installed at individual chillers and header/risers)
- Chiller(s) part load performance (at 10% interval from 100% to minimum value) from equipment supplier at operating conditions.

5.5 Registration and Renewal of Registration of Energy Auditor - Qualifications and Experience

- **5.5.1** An individual shall be eligible to be registered as an Energy Auditor with BCA under Section 22FG of the BC Act if he holds the following qualifications:
 - Possess at least an engineering or a building-related undergraduate degree in architecture, building science, facility management, or sustainable building design, or any equivalent professional qualification acceptable by the Commissioner of Building Control; and

- ii. Have at least 3 years of relevant practical experience in central airconditioning design and installation, or operation; and
- iii. Completed 2 ASHRAE Level III Energy Audits or 3 Periodic Energy Audits on Building Cooling Systems under the supervision of a PE(Mechanical) or BCA registered Energy Auditor respectively; and
- iv. Passed the interview and/or exam by the Energy Auditor Committee.
- 5.5.2 For the purpose of ascertaining whether an individual is capable of carrying out the duties of an Energy Auditor, the Commissioner of Building Control shall decide based on the recommendations provided by the Energy Auditor Registration Committee appointed by him. An application to be registered as an Energy Auditor can be made using the Application Form BCA-EAS-APL which can be either obtained in hardcopy forms or downloaded from http://www.bca.gov.sg. All applications must be accompanied with a fully completed form, supporting certificates and evidence of related work experiences as specified in the form.
- 5.5.3 The registration as an Energy Auditor shall be valid for 3 years, and each renewal of the registration shall also be valid for 3 years. Application for renewal of registration can be made using the Application Form BCA-EAS-RAPL which can either be obtained in hardcopy forms or downloaded from http://www.bca.sg. All applications must be accompanied with a fully completed form, supporting certificates and evidence of related work experiences as specified in the form and renewal fee.

6. ENERGY AUDIT METHODOLOGY FOR CHILLERS

6.1 Minimum OSE Standards for Energy Audit Submission

6.1.1 This section is for buildings that have undergone major energy-use change under Part IIIB of the BC Act and are required to comply with the Code on Environmental Sustainability Measures for Existing Buildings.

Water Cooled Chilled-Water Plant

	Building Cod	oling Load
Baseline	< 500RT ≥500RT	
Minimum OSE (kW/RT)	0.80*	0.75

^{*} For project with Design Score approved before 1 Apr 2019, the minimum OSE is 0.85kW/RT

Air Cooled Chilled-Water Plant/ Unitary Air conditioners

Baseline	Building Cooling Load				
	< 500RT	≥500RT			
Minimum OSE(kW/RT)	1.1	1.0			

- 6.1.2 This section is for buildings subject to the Building Control (Environmental Sustainability) Regulations 2008 and where the applications for planning permission were submitted to the competent authority under the Planning Act (Cap. 232) on or after 1st December 2010.
 - 6.1.2.1 This subsection shall apply to the whole or part of any building that is on land within areas stipulated under the First and Second Schedules to the 2008 Regulations (and such other schedules to the 2008 Regulations as gazetted from time to time). The minimum OSE standard is as listed below and is based on requirements listed in the First and Second Schedules to the 2008 Regulations (and such other schedules to the 2008 Regulations as gazetted from time to time).

Water Cooled Chilled-Water Plant

Baseline	Building Cooling Load			
Daseille	< 500RT	≥500RT		
Minimum OSE(kW/RT) - Gold ^{plus} - Platinum	0.7 0.7	0.65 0.65		

Air Cooled Chilled-Water Plant

	Building Co	oling Load		
Baseline	< 500RT	≥500RT		
Minimum OSE(kW/RT) - Gold ^{plus} - Platinum	0.85 0.78	Not applicable		

6.1.2.2 This subsection shall apply to all other buildings. The minimum OSE standard is as listed below.

Water Cooled Chilled-Water Plant

	Building Cooling Load				
Baseline	< 500RT	≥500RT			
Minimum OSE(kW/RT)	0.8	0.7			

Air Cooled Chilled-Water Plant

	Building Cooling Load				
Baseline	< 500RT	≥500RT			
Minimum OSE(kW/RT)	0.9	0.8			

- **6.1.3** Where there is a combination of water cooled and air cooled Chilled Water Plant, the minimum OSE shall be based on the weighted average of the air conditioning system.
- **6.1.4** The Energy Audit shall be carried out over the normal operating hours for the various categories of buildings as defined below. The data required to establish the OSE shall be sampled and acquired simultaneously and continuously for a minimum of one (1) week at one (1) minute interval.

Office Buildings:

 Monday to Friday: 9am to 6pm

Retail Malls:

Monday to Sunday: 10am to 9pm

Hotels/Hospitals:

Monday to Sunday: 24 hours

Other building types:

 To be determined based on the operating hours

6.2 Heat Balance Substantiating Test

6.2.1 Verification of the OSE shall be by computing the system heat balance of the water-cooled Chilled Water Plant in accordance with SS591 to the extent as prescribed.

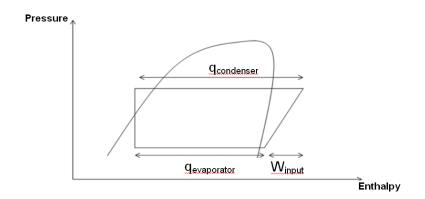
Note: For air-cooled Chilled Water Plant, heat balance requirements will not be applicable.

- **6.2.2** The heat balance shall be computed over the entire normal operating hours as defined in clause 6.1.4 with more than 80% of the computed heat balance within 5% over a one (1) week period.
- **6.2.3** For a perfectly balanced chiller system, the heat balance can be represented by,

$$q_{condenser} = q_{evaporator} + W_{input}.$$

where $q_{condenser} = heat \ rejected \ by \ condenser, \ kW$
 $q_{evaporator} = heat \ gain \ in \ evaporator, \ kW$
 $W_{input} = power \ input \ to \ compressor, \ kW$

6.2.4 The pressure enthalpy diagram below shows the concept of a heat balance equation in a vapour compression cycle.



Pressure Enthalpy Chart

6.2.5 The system heat balance of the Chilled Water Plant shall be computed using the formula stated below over the normal operating hours,

6.2.6 For open drive chillers, the W_{input} shall take into account the motor efficiency provided by the manufacturer. An example is provided as follows:

Motor rated efficiency (
$$\eta$$
) = 90%

Adjusted power input to compressor
$$W_{input} = 100 \text{ kW x } 90\%$$

= 90 kW

- **6.2.7** Where hydraulic losses of pumps constitute a substantial heat gain, these losses could be accounted for. The values shall be determined from motor efficiency and pump efficiency values provided by the manufacturer. Examples are illustrated as follows:
 - (a) For chilled water pump(s) adjustment,

Motor input power (measured) = 30 kW (A)
Motor rated efficiency (
$$\eta$$
) = 90% (B)

Pump rated efficiency (
$$\eta$$
) = 80% (C)

Hydraulic losses¹ = (A) x (B) x [(
$$100\%$$
 – (C)]

$$= 5.4 kW$$

Adjusted total input power
$$W_{input}$$
 = kW_i (chillers) + 5.4kW

Where kW_i (chillers) = adjusted power input to compressor, kW

(b) For condenser water pump(s) adjustment,

Motor input power (measured) =
$$20 \text{ kW}$$
 (A)

Motor rated efficiency (η) = 90% (B) Pump rated efficiency (η) = 80% (C)

Hydraulic losses¹ = (A) x (B) x [(100% - (C)]]

= 20kW x 90% x (100% - 80%)

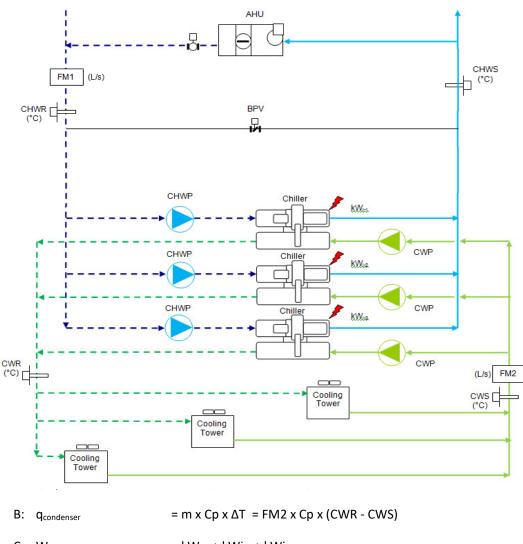
= 3.6 kW

Adjusted $q_{condenser(adj)}$ = $q_{condenser}$ - 3.6kW

¹ For equipment with VSD, the equation shall include variable speed drive losses, harmonic filter losses (where applicable).

ANNEX A - Determining Heat Balance for Different Plant Configuration

Plant A - Constant Primary Chilled-Water System



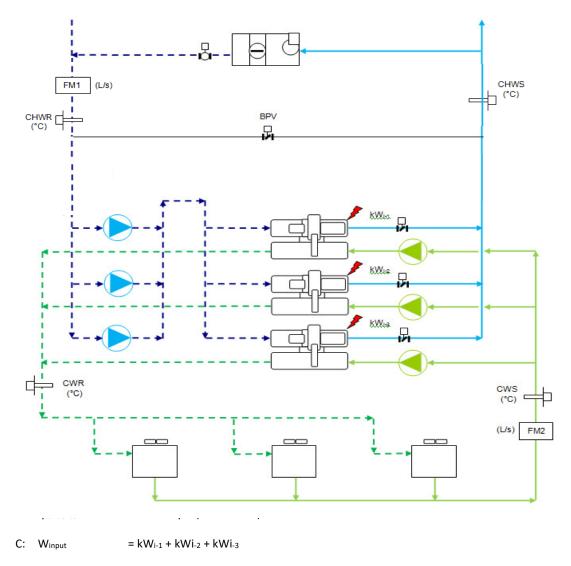
C: W_{input} = $kW_{i-1} + kWi_{-2} + kWi_{-3}$

where Cp = 4.19 kJ/kg °C & density of water is assumed to be 1kg/L

Percent heat balance = $[(A + C) - B] / B \times 100\%$

Note: Hydraulic losses of pumps constituting substantial heat gain, $W_{input}/q_{condenser}$ may be adjusted to account for these additional heat gains. The values shall be determined from variable speed drive losses, harmonic filter losses (where applicable), motor efficiency and pump efficiency values certified by the manufacturer.

Plant B - Variable Primary Chilled-Water System

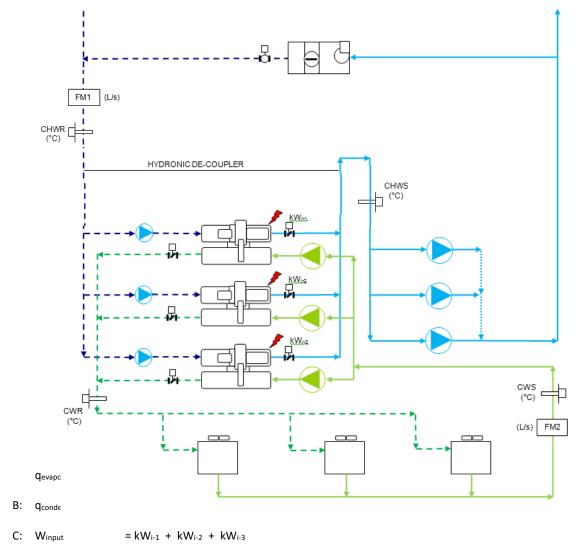


where Cp = 4.19 kJ/kg °C & density of water is assumed to be 1 kg/L

Percent heat balance = $[(A + C) - B] / B \times 100\%$

Note: In the event where hydraulic losses of pumps constitute substantial heat gain, $W_{input}/q_{condenser}$ may be adjusted to account for these additional heat gains. The value shall be determined from variable speed drive losses, harmonic filter losses (where applicable), motor efficiency and pump efficiency values certified by the manufacturer.

Plant C - Constant Primary & Variable Secondary Chilled-Water System



where Cp = 4.19 kJ/kg °C & density of water is assumed to be 1kg/L

Percent heat balance = $[(A + C) - B] / B \times 100\%$

Note: In the event where hydraulic losses of pumps constitute a substantial heat gain, these losses have to be properly accounted for. The value shall be determined from variable speed drive losses, harmonic filter losses (where applicable) and pump efficiency values certified by the manufacturer.

The following example illustrates a successful heat balance where 80% of the computed heat balance falls within \pm 5% as required.

	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)
	Chilled water supply temperature	Chilled water return temperature	Chilled water flow rate	Condenser water supply temperature	Condenser water return temperature	Condenser water flow rate	Chiller kWe	Heat Gain	Heat Rejected	Percent Heat Balance
dd/mm/yyyy hh:mm	ပ	°C	L/s	့	့	L/s	kW	RT	RT	%
16/6/2012 15:00	6.70	12.60	84.10	29.4	35.5	97.65	308	591.14	709.65	-4.36
16/6/2012 15:01	6.71	12.50	84.20	29.5	35.4	97.60	309	580.81	686.03	-2.53
16/6/2012 15:02	6.72	12.30	84.30	29.6	35.3	97.55	310	560.41	662.44	-2.10
16/6/2012 15:03	6.73	12.10	84.20	29.7	35.2	97.50	311	538.68	638.86	-1.84
16/6/2012 15:04	6.74	12.20	84.10	29.8	35.1	97.55	312	547.05	615.95	3.22
16/6/2012 15:05	6.75	12.00	84.00	29.9	35	97.60	311	525.39	593.01	3.51
16/6/2012 15:06	6.74	12.30	84.10	29.8	35.1	97.65	310	557.07	616.58	4.64
16/6/2012 15:07	6.73	12.10	84.20	29.7	35.2	97.60	309	538.68	639.52	-2.03
16/6/2012 15:08	6.72	12.10	84.30	29.6	35.3	97.55	308	540.32	662.44	-5.21
16/6/2012 15:09	6.71	12.20	84.20	29.5	35.4	97.50	309	550.71	685.33	-6.82
16/6/2012 15:10	6.70	12.40	84.10	29.4	35.2	97.55	310	571.10	674.06	-2.20
16/6/2012 15:11	6.70	12.60	84.10	29.4	35.5	97.65	308	591.14	709.65	-4.36
16/6/2012 15:12	6.71	12.50	84.20	29.5	35.4	97.60	309	580.81	686.03	-2.53
16/6/2012 15:13	6.72	12.30	84.30	29.6	35.3	97.55	310	560.41	662.44	-2.10
16/6/2012 15:14	6.73	12.10	84.20	29.7	35.2	97.50	311	538.68	638.86	-1.84
16/6/2012 15:15	6.74	12.20	84.10	29.8	35.1	97.55	312	547.05	615.95	3.22
16/6/2012 15:16	6.75	12.00	84.00	29.9	35	97.60	311	525.39	593.01	3.51
16/6/2012 15:17	6.74	12.30	84.10	29.8	35.1	97.65	310	557.07	616.58	4.64
16/6/2012 15:18	6.73	12.10	84.20	29.7	35.2	97.60	309	538.68	639.52	-2.03
16/6/2012 15:19	6.72	12.10	84.30	29.6	35.3	97.55	308	540.32	662.44	-5.21
16/6/2012 15:20	6.71	12.20	84.20	29.5	35.4	97.50	309	550.71	685.33	-6.82
16/6/2012 15:21	6.70	12.40	84.10	29.4	35.2	97.55	310	571.10	674.06	-2.20
Total							6814	12,202.71	14,367.72	32.36
	Total data count				22					
	Data Count > +5% error					0				
	Data Count < -5% error					4				
	Percentage of heat balance within ± 5%						82%			

Heat Gain (h) = $m \times Cp \times \Delta T = (c) \times 4.19 \text{kJ/kg} \, ^{\circ}\text{C} \times [(b) - (a)] / 3.517$

Heat Rejected (i) = (f) x 4.19 kJ/kg $^{\circ}$ C x [(e) – (d)] / 3.517

Percent Heat Balance (j) = $100 \times [(g) / 3.517 + (h) - (i)] / (i)$

ANNEX B - Energy Audit Report For Building Cooling System

Date: DD/MM/YYYY

ENERGY AUDIT REPORT FOR BUILIDNG COOLING SYSTEM

FOR

ENTER BUILDING NAME

At

ENTER BUILDING ADDRESS

(BUILDING IMAGE)

Submitted By

Enter name of PE/Energy Auditor

Signature of PE/Energy Auditor

PE (Mech) Registration No*: Enter No.

Energy Auditor Registration No*: Enter No.

*Delete whichever is not applicable

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*required if using wet hulb temperature as set point

1.0 Executive Summary & Recommendation

(Example)

This report highlights the findings and recommendations obtained from the energy audit performed at <u>Enter Building Name</u> from [Enter Period of Audit] DD/MM/YYYY to DD/MM/YYYY for 24 hrs.

Corrective measures taken by PE (Mech)/ Energy Auditor to comply with PEA Notice.

- 1) <Description of findings/ measures>
- 2) <Description of findings/ measures>
- 3) <Description of findings/ measures>

Recommended energy improvement measures for Building Owners:

- 1) <Description of recommendations>
- 2) < Description of recommendations>
- 3) <Description of recommendations>

2.0 Building Information

Enter a brief description of the building here.

Project Reference Number : <u>Enter project reference indicated in CORENET submission</u>

Building Name :

Building Address :

Postal Code :

Building Type :

Building Age :

Date of last Energy Audit Submission :

Gross floor area (GFA), m² :

Air conditioned area, m² :

Number of guest rooms :

(for hotels/service apartments)

3.0 Energy Audit Information For Building Cooling System

Enter PE(Mechanical) / Energy Auditor Name was appointed by Enter Owner Name/ MCST, owner of Enter Building Name to be the Energy Auditor for the 3 yearly submission of the operating system efficiency (OSE) of the centralized Chilled Water Plant. The report will present the performance of centralized Chilled Water Plant efficiency based on the measurements from the permanent instrumentations installed on site.

Location : Enter location of Chilled Water Plant

Energy Audit Period : <u>Enter Energy Audit period</u>

*Note: Minimum 1 week

Date of notice served : Enter date of notice served by BCA

Date of submission in notice : <u>Enter submission deadline stipulated in BCA notice</u>

OSE standard to comply (kW/RT): Enter Min OSE standard for Chilled Water Plant

(GLS/ non-GLS)

Data Logging Interval : 1 minute sampling

Trend Logged Parameters* : Chilled Water Supply main header temperature

Chilled Water Return main header temperature

Chilled Water flow rate at chilled water return main

header

Condenser Water Supply main header temperature

Condenser Water Return main header temperature

Condenser water flow rate at condenser water

return main header

Power input to Chiller(s)

Power input to Chilled water pump(s)

Power input to Condenser water pump(s)

Power input to Cooling tower(s)

^{*} Trend logged parameters are not limited to the above and may vary depending on the piping and electrical circuit design.

3.1 Chilled Water Plant Design information*

ID	Description	Brand	Туре	Name plate motor (kW)	Total Cooling Capacity (RT)	Chilled water LWT/EWT	Rated Efficiency kW/RT	Year Installed
CH01	Chiller 1	Brand X	Centrifugal, water- cooled	162.8	300	7.5 °C	0.543	2017
CH02	Chiller 2	Brand X	Centrifugal, water- cooled	162.8	300	7.5 °C	0.543	2017
CH02	Chiller3	Brand X	Centrifugal, water- cooled	162.8	300	7.5 °C	0.543	2017

Table 1: Chiller Information (Example)

ID	Brand	Туре	Name plate motor (kW)	Pump Head (m)	Flow rate (L/S)	Rated Pump/ Fan efficiency	Rated Motor Efficiency
CHWP 1	Brand X	end suction	11	23	33.65	80.0%	92.4%
CHWP 2	Brand X	end suction	11	23	33.65	80.0%	92.4%
CHWP 3	Brand X	end suction	11	23	33.65	80.0%	92.4%
CWP 1	Brand Y	end suction	15.0	16.0	56.82	79.0%	92.4%
CWP 2	Brand Y	end suction	15.0	16.0	56.82	79.0%	92.4%
CWP 3	Brand Y	end suction	15.0	16.0	56.82	79.0%	92.4%
CT 1	Brand Z	cross flow	5.5 x 1 Cell		66.2	75%	86%
CT 2	Brand Z	cross flow	5.5 x 1 Cell	-	66.2	75%	86%
CT 3	Brand Z	cross flow	5.5 x 1 Cell	-	66.2	75%	86%

Table 2: Ancillary equipment Information (Example)

^{*}Based on equipment design specifications and name plate ratings

3.2 Chilled Water Plant Normal Operating Hours

Monday to Friday : 1000 – 2100 Hrs

Saturday / Sunday : No operations

Note: The operating hours should follow the table in clause 6.1.4

3.3 Description of Plant Control Strategy

Summary of the present plant control strategy adopted for the applicant's building chiller plant systems' operation. You may include but not limited to the following:

1) Chiller sequencing

Describe how the chiller(s) operate to handle the varying building cooling load e.g. chiller cut-in/out sequence varying with building load and addressing peak and off peak load based on (supply water temperature, and/or building load, and/or compressor current running load amps) and time delay.

2) Chilled water pump (if applicable)

Describe the parameters used to control chilled water pumps
e.g. pump speed modulate based on ((differential) pressure sensor located at chiller
header, or remote AHU cooling coil, or several zones of AHU cooling coil, or
optimising pump pressure by critical valve control), set-point(s) and bypass valve
controls to ensure chillers operate at minimum flow rate

3) Condenser water pump (if applicable)

Describe the parameters used to control condenser water pumps e.g. modulate to maintain condenser water differential temperature set point or gpm/ton and the set-point(s).

4) Cooling tower (if applicable)

Describe the parameters used to control cooling towers

e.g. Modulate base on cooling tower approach temperature (difference between CT leaving water temperature and ambient wet-bulb temperature) set point (adjustable), or scheduled cooling tower leaving temperature set point, or dynamic optimized cooling tower leaving water temperature set point and the set-point(s)

5) Other optimisation (if applicable)

Describe any other optimisation used

e.g. Chilled water supply temperature reset. At off-peak period, reset based on outdoor air temperature/humidity, or VPF bypass control, or predefined schedule. (Note: Resetting CHW temperature may incur higher pump power and may compromise on space temperature and relative humidity)

(Example)

Chiller Configuration: <x> unit(s) of <x> RT chiller & <x> unit(s) of <x> RT chiller

Variable Primary Chilled Water System

Variable Condenser Pump

1) Chiller sequencing

Scenario for Cut-in: Chilled water supply header temperature is above set point of <x> $^{\circ}$ C + <deadband> \underline{OR} total system tonnage is above <x> RT for a period of <x> minutes.

Scenario for Cut-out: Chilled water supply header temperature is below set point of <x> °C + <deadband> <u>AND</u> total system tonnage is below <x> RT for a period of <x> minutes.

Time delay: Whenever any chiller cuts-in/out, there is <x> minutes delay to allow system to stabilize.

2) Chilled water pump (CHWP)

Primary CHWP speed is modulated to maintain a differential pressure set point of $\langle x \rangle$ psi + \langle deadband \rangle . Differential pressure sensors are installed at chilled water pipe headers. CHWP speed is limited to $\langle x \rangle$ Hz to ensure chillers running at minimum flow. When CHWP speed ramps down to minimum and differential pressure rises above set point, the bypass valve will open to maintain DP set point and minimum flow rate.

3) Condenser water pump (CWP) <fixed/variable>

Minimum running speed of CWP is <x> Hz. When condenser flow rate is reduced to set point of <x> l/s or <x> gpm/ton, CWP speed would be increased and vice versa.

4) Cooling Tower (CT)

CT fan speed is modulated to maintain leaving condenser water temperature set point of <x> °C which is equal to outdoor air wet-bulb temp plus <x> °C. When chiller(s) is in operation, all CTs would be turn on. When CT leaving water temperature falls below the set point, CT fan speed would be decreased until minimum speed of <x> Hz.

5) Other Optimisation

Chilled water temperature set point is reset to <x> °C during off-peak period from 2000hrs to 0800hrs.

4.0 Instrumentations

Accurate measuring instruments complying with the Code on Environmental Sustainability Measures for Existing Buildings or the Code for Environmental Sustainability of Buildings (2nd edition and onwards) that is prevailing at the time of installation were used during the audit to gather information on the power consumption, temperatures and flow rate.

The points of measurements are listed in the following table:

ID / Serial No.	Brand	Sensor Type	Installation Location	Measurement/ Calibration range	Measurement Uncertainty (%)	Last Calibration Date	Calibration Laboratory
EP80367	Brand X	10K Ω Thermistor	CHWS Header	0.01°C – 29.765°C	±0.03 °C	09/05/2014	XX laboratory
EP80364	Brand X	10K Ω Thermistor	CHWR Header	0.01°C – 29.765°C	±0.03 °C	09/05/2014	XX laboratory
EP80361	Brand X	10K Ω Thermistor	CWS Header	0.01°C – 29.765°C	±0.03 °C	09/05/2014	XX laboratory
EP80363	Brand X	10K Ω Thermistor	CWR Header	0.01°C – 29.765°C	±0.03 °C	09/05/2014	XX laboratory
3k672013 43004	Brand X	Magnetic Full Bore	CHWR Header	0 l/s- 288.63 l/s	0.5%	29/10/2013	factory calibration
3k672014 18063	Brand X	Magnetic Full Bore	CWR Header	0 l/s- 483.33 l/s	0.5%	09/05/2014	factory calibration
38498	Brand X	True RMS, 3 phase	MSB Incoming 1	60 – 600 kW	0.5%	08/07/2014	factory calibration
1402404	Brand X	True RMS, 3 phase	MSB Incoming 2	60 – 600 kW	0.5%	08/07/2014	factory calibration
38491	Brand X	True RMS, 3 phase	CH/6-1	60 – 300 kW	0.5%	08/07/2014	factory calibration
38487	Brand X	True RMS, 3 phase	CHP/6-1	0 – 30 kW	0.5%	08/07/2014	factory calibration
38490	Brand X	True RMS, 3 phase	CWP/6-1	0 – 30 kW	0.5%	08/07/2014	factory calibration
38499	Brand X	True RMS, 3 phase	CT/6-1	0 – 30 kW	0.5%	08/07/2014	factory calibration
38497	Brand X	True RMS, 3 phase	CH/6-2	0 – 300 kW	0.5%	08/07/2014	factory calibration
38483	Brand X	True RMS, 3 phase	CHP/6-2	0 – 30 kW	0.5%	08/07/2014	factory calibration
1402325	Brand X	True RMS, 3 phase	CWP/6-2	0 – 30 kW	0.5%	08/07/2014	factory calibration
38572	Brand X	True RMS, 3 phase	CT/6-2	0 – 30 kW	0.5%	08/07/2014	factory calibration
1402399	Brand X	True RMS, 3 phase	CH/6-3	60 – 300 kW	0.5%	08/07/2014	factory calibration
38574	Brand X	True RMS, 3 phase	CHP/6-3	0 – 30 kW	0.5%	08/07/2014	factory calibration
38485	Brand X	True RMS, 3 phase	CWP/6-3	0 – 30 kW	0.5%	08/07/2014	factory calibration
38486	Brand X	True RMS, 3 phase	CT/6-3	0 – 30 kW	0.5%	08/07/2014	factory calibration

Table 3: Instrumentation Table (Example)

5.0 Chiller Plant Performance Analysis (1 week data)

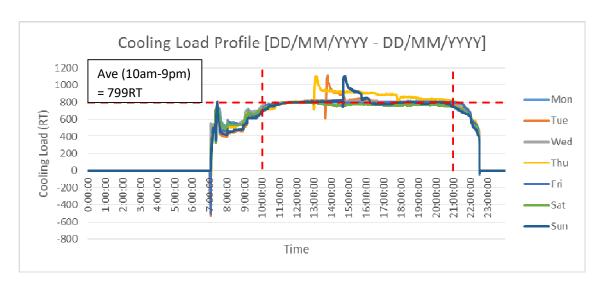


Figure 1: Super-imposed plot of 24 hr Cooling Load Profile RT (Example)

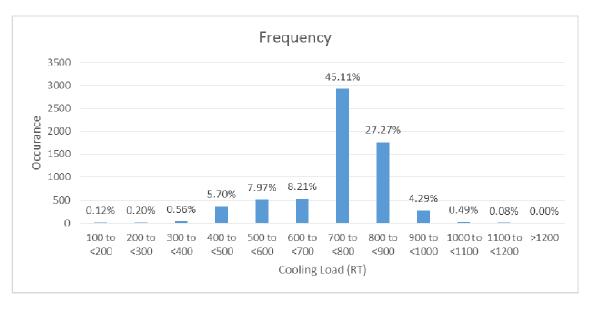


Figure 2: Histogram of Cooling Load Occurrences (Example)

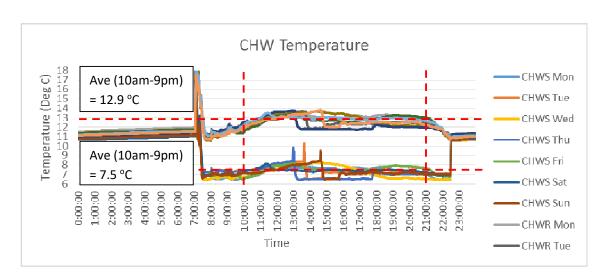


Figure 3: Super-imposed plot of daily chilled water supply/return temperature °C (Example)

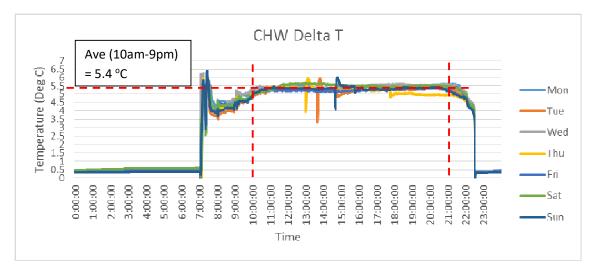


Figure 4: Super-imposed plot of daily chilled water temperature difference °C (Example)

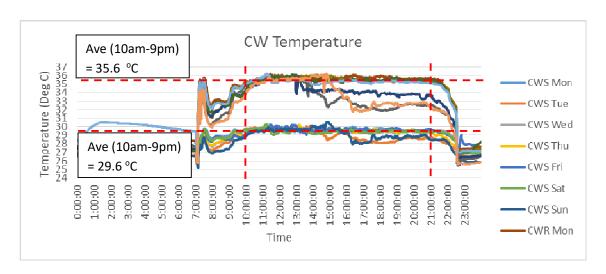


Figure 5: Super-imposed plot of daily condenser water supply/return temperature °C (Example)

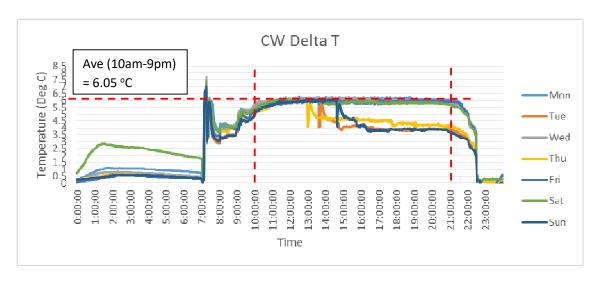


Figure 6: Super-imposed plot of daily condenser water temperature difference °C (Example)

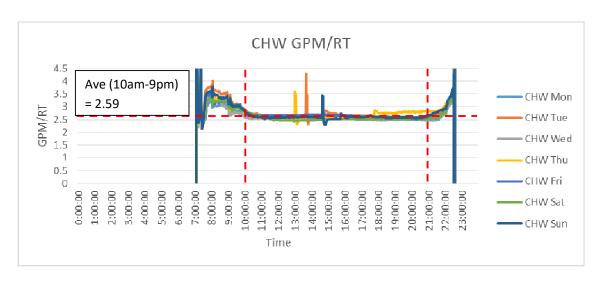


Figure 7: Super-imposed plot of daily chilled water GPM/RT (Example)

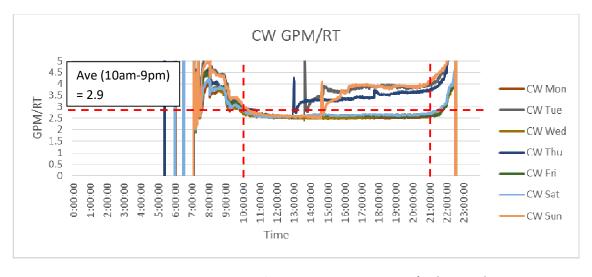
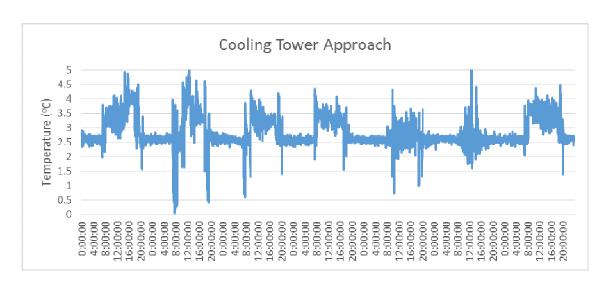


Figure 8: Super-imposed plot of daily condenser water GPM/RT (Example)



*Figure 9: Cooling Tower Approach Temperature (Example)

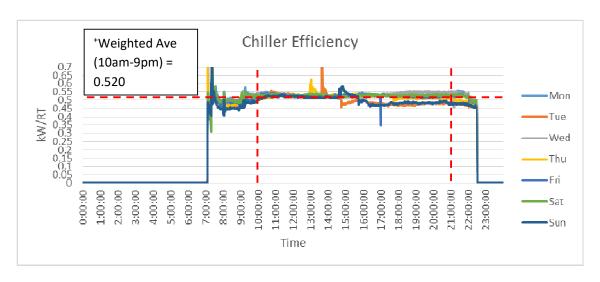


Figure 10: Super-imposed plot of daily chiller efficiency kW/RT (Example)

^{*}required if using wet bulb temperature as set point

*Weighted average: ∑ kW-hr / ∑ RT-hr

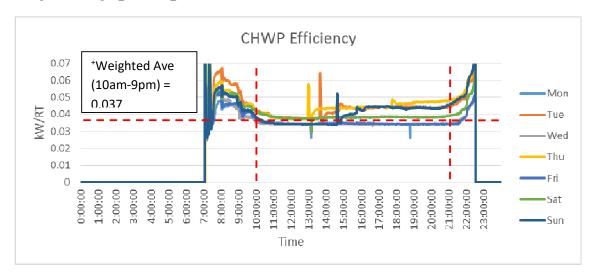


Figure 11: Super-imposed plot of daily chilled water pump efficiency kW/RT (Example)

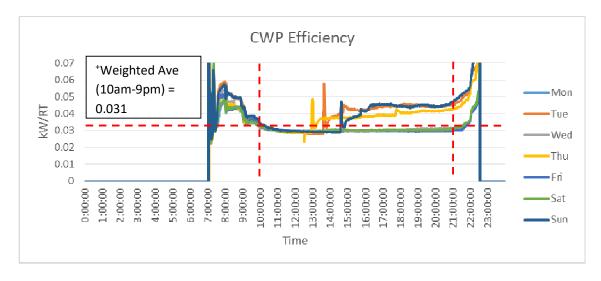


Figure 12: Super-imposed plot of daily condenser water pump efficiency kW/RT (Example)

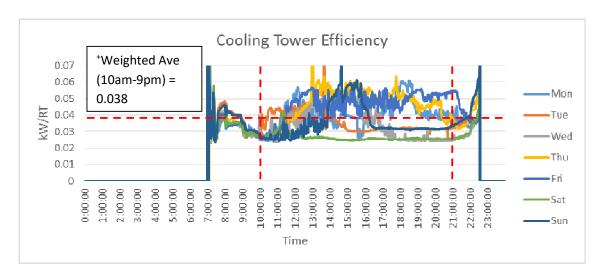


Figure 13: Super-imposed plot of daily cooling tower efficiency kW/RT (Example)

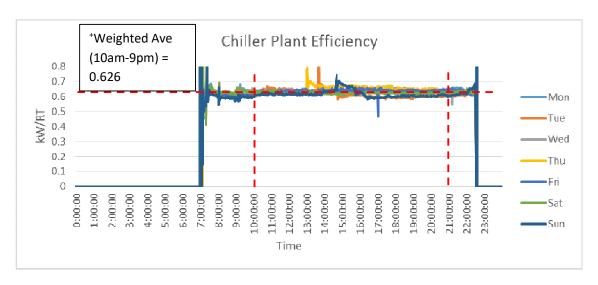


Figure 14:Super-imposed plot of daily chiller plant system efficiency kW/RT (Example)

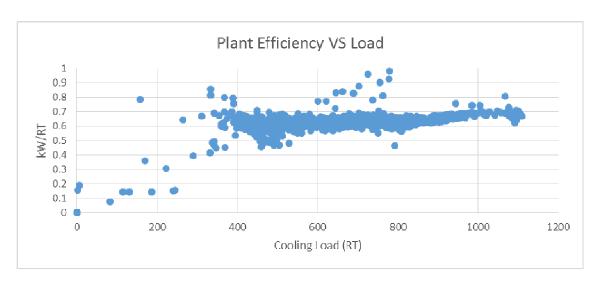


Figure 15: Scatter plot of chiller plant efficiency over cooling load (Example)

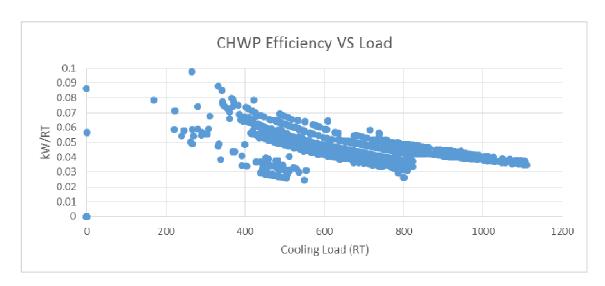


Figure 16: Scatter plot of chilled water pump efficiency over cooling load (Example)

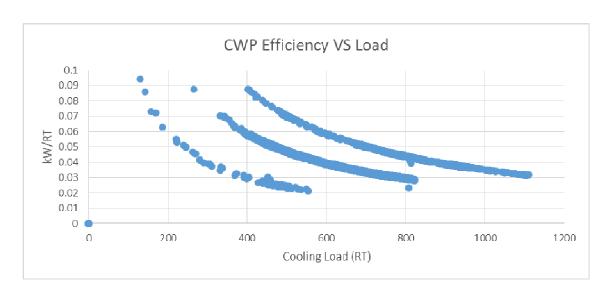


Figure 17: Scatter plot of condenser water pump efficiency over cooling load (Example)

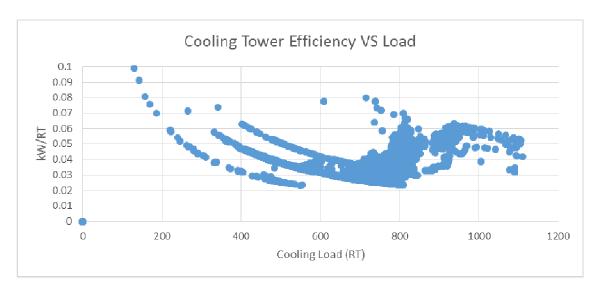


Figure 18: Scatter plot of cooling tower efficiency over cooling load (Example)

5.1 Summary of Chilled Water Plant Operating Performance

Daily Average Reading	Pe	Unit	
Daily Average Reducing	Daytime^	Night-time~	
Cooling Load			RT
Cooling Load Density (Air-con area)			m2/RT
Power Consumption			kW
Chilled water supply temperature			°C
Chilled water return temperature			°C
Chilled water delta T			°C
Chilled water flow rate			I/s
Chilled water flow rate vs cooling load			USgpm/RT
*Condenser heat rejection			HRT
*Condenser water supply temperature			°C
*Condenser water return temperature			°C
*Condenser water delta T			°C
*Condenser water flow rate			I/s
*Condenser water flow rate vs cooling load			USgpm/RT
Chiller(s) efficiency			kW/RT
Chilled water pump(s) efficiency			kW/RT
*Condenser water pump(s) efficiency			kW/RT
*Cooling tower(s) efficiency			kW/RT
Overall chiller plant efficiency			kW/RT

Table 4: Chilled Water Plant Performance Summary

^{*}Not applicable to air-cooled Chilled Water Plant

[~]For hotels and other developments with 24-hour operations only; Night-time shall refer to the period from 11pm – 7am;

[^] For hotels and other developments with 24-hour operations, day-time shall refer to the period from 7am – 11pm; for all other developments, daytime shall refer to the normal operating hours stipulated in clause 6.1.4 of the PEA code

6.0 Summary of Heat Balance

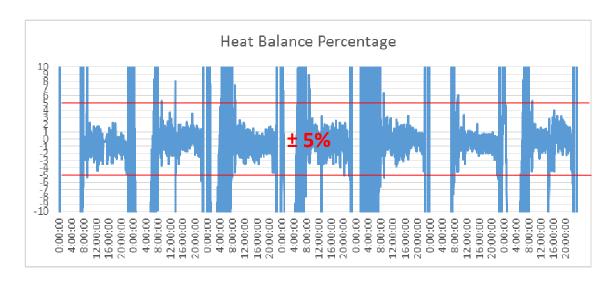


Figure 19: System Level Heat Balance Plot (Example)

	Quantity	Unit	Formula
Sum of total electrical energy used		kWh	(A)
Sum of total cooling produced		RTh	(B)
Sum of total heat rejected		RTh	(C)
Chiller Plant Efficiency		kW/RT	(A) / (B)
Total Heat Balance Data Count		-	(D)
Data Count > + 5% error		-	(E)
Data Count < - 5% error		-	(F)
Data Count within ±5% error		-	(G) = (D) - (E) - (F)
% Heat Balance within ±5% error		%	100 x (G) / (D)

Table 5: Heat Balance Summary

7.0 Schedule of space operating conditions

(10 points Spot measurements)

		Normal opera conditi	ting room ons	Measured		
	Room name (i.e. Air conditioned occupied/ common Spaces)	Dry Bulb Temperature (°C)	Relative Humidity (%)	*Dry Bulb Temperature (°C)	*Relative Humidity (%)	*CO2 Concentration (ppm)
1	i.e. Office 1					
2	i.e. Office 2					
3	i.e. Meeting Room 1					
4	i.e. Meeting Room 2					
5						
6						
7						
8						
9						
10						

Table 6: Space Condition Schedule (Example)

^{*} Any observation on over-cooling/ under-cooling and ventilating of space conditions should first be investigated and corrected before the energy audit is carry out. Refer to recommended limits of SS553 and SS 554.

APPENDIX A

Checklist of Plant Operating Condition (for best practices)

	Yes	No	Actual value
Is the airside efficiency ≤ 0.2 kW/RT?			
Is Chilled water delta T >5.5 °C?			
Is the cooling tower approach temperature ≤ 2.0 °C as compared with outdoor wet bulb temperature?			
Is the Chilled water pump efficiency ≤ 0.03 kW/RT?			
Is the Condenser water pump efficiency ≤ 0.035 kW/RT?			
Is the Cooling Tower efficiency ≤ 0.03 kW/RT?			
Does Refrigerant Condenser approach within the range of 0.5 °C to 1.5 °C?			
Does Refrigerant Evaporator approach within the range of 0.5 °C to 1.5 °C?			

Table 7: Checklist of Plant Operating Condition (for best practices)

APPENDIX B

Temperature Sensor Verification Plot (worked example)

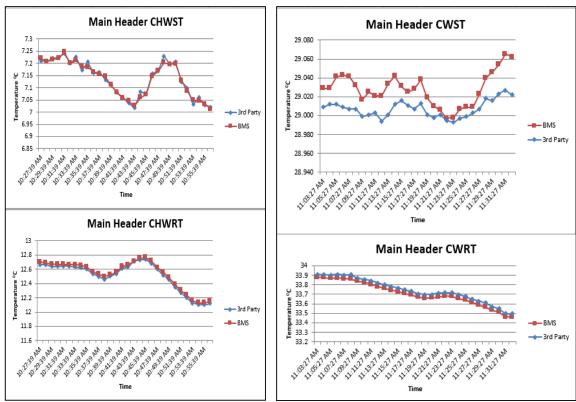


Figure 20: Temperature Verification Plots for Water-Cooled Chiller Plant System (Example)

Project Nam	ie: <u>Enter nam</u> e	e of project													
Date of Veri	fication: Enter	date (dd/mi	m/yyyy)												
Verification	by: Enter nam	e of PE (Med	h)/Energy Au	uditor											
Main Header Chilled Water Supply Temperature			Main Header Chilled Water Return Temperature			Main Header Condenser Water Supply Temperature			Main Header Condenser Water Return Temperature						
Time	3rd party (°C)	BMS (°C)	ABS	Time	3rd party (°C)	BMS (°C)	ABS	Time L:03:27 AM	3rd party (°C) 29.009	BMS (°C) 29.029	-0.020	Time 11:03:27 AM	3rd party (°C) 33.912	BMS (°C) 33.879	ABS 0.033
10:27:39 AM	7.211	7.220	-0.009	10:27:39 AM	12.663	12.703	-0.040	L:04:27 AM	29.012	29.029	-0.017	11:04:27 AM	33.907	33.876	0.031
10:28:39 AM	7.209	7.207	0.002	10:28:39 AM	12.665	12.690	-0.025	L:05:27 AM	29.012	29.041	-0.029	11:05:27 AM	33,904	33.867	0.037
10:29:39 AM	7.221	7.216	0.005	10:29:39 AM	12.640	12.674	-0.034	L:06:27 AM	29.009	29.043	-0.034	11:06:27 AM	33.910	33.874	0.036
10:30:39 AM	7.225	7.220	0.005	10:30:39 AM	12.640	12.671	-0.031	L:07:27 AM	29.007	29.041	-0.034	11:07:27 AM	33.902	33.864	0.038
10:31:39 AM	7.240	7.246	-0.006	10:31:39 AM	12.642	12.668	-0.026	L:08:27 AM	29.007	29.032	-0.025	11:08:27 AM	33.906	33.862	0.044
10:32:39 AM	7.200	7.200	0.000	10:32:39 AM	12.642	12.663	-0.021	1:09:27 AM	28,999	29.017	-0.018	11:09:27 AM	33.867	33.840	0.027
10:33:39 AM	7.227	7.211	0.016	10:33:39 AM	12.628	12.662	-0.034	1:10:27 AM	29.001	29.025	-0.024	11:10:27 AM	33.858	33.826	0.032
10:34:39 AM	7.172	7.186	-0.014	10:34:39 AM	12.622	12.653	-0.031	1:11:27 AM	29.003	29.021	-0.018	11:11:27 AM	33.841	33.806	0.035
10:35:39 AM	7.205	7.184	0.021	10:35:39 AM	12.601	12.631	-0.030	L:12:27 AM	28.994	29.021	-0.027	11:12:27 AM	33.818	33.781	0.037
10:36:39 AM	7.160	7.164	-0.004	10:36:39 AM	12.537	12.567	-0.030	L:13:27 AM	29.001	29.034	-0.033	11:13:27 AM	33.801	33.764	0.037
10:37:39 AM	7.164	7.154	0.010	10:37:39 AM	12.499	12.532	-0.033	-:14:27 AM	29.012	29.042	-0.030	11:14:27 AM	33.780	33,744	0.036
10:38:39 AM	7.133	7.146	-0.013	10:38:39 AM	12.460	12.497	-0.037	L:15:27 AM	29.016	29.031	-0.015	11:15:27 AM	33.767	33.729	0.038
10:39:39 AM	7.111	7.111	0.000	10:39:39 AM	12.501	12.523	-0.022	L:16:27 AM	29.011	29.025	-0.014	11:16:27 AM	33.746	33.709	0.037
10:40:39 AM	7.079	7.080	-0.001	10:40:39 AM	12.535	12.560	-0.025	L:17:27 AM	29.007	29.028	-0.021	11:17:27 AM	33.734	33.696	0.038
10:41:39 AM	7.059	7.057	0.002	10:41:39 AM	12.614	12.640	-0.026	L:18:27 AM	29.013	29.038	-0.025	11:18:27 AM	33.706	33.672	0.034
10:42:39 AM	7.037	7.046	-0.009	10:42:39 AM	12.631	12.664	-0.033	L:19:27 AM	29.001	29.019	-0.018	11:19:27 AM	33.697	33.660	0.037
10:43:39 AM	7.019	7.026	-0.007	10:43:39 AM	12.712	12.710	0.002	L:20:27 AM	28.998	29.010	-0.012	11:20:27 AM	33.698	33.662	0.036
10:44:39 AM	7.082	7.060	0.022	10:44:39 AM	12.732	12.758	-0.026	1:21:27 AM	29.001	29.006	-0.005	11:21:27 AM	33.712	33.670	0.042
10:45:39 AM	7.076	7.073	0.003	10:45:39 AM	12.736	12.767	-0.031	L:22:27 AM	28.995	28.997	-0.002	11:22:27 AM	33.718	33.681	0.037
10:46:39 AM	7.157	7.147	0.010	10:46:39 AM	12.683	12.718	-0.035	L:23:27 AM	28.993	28.998	-0.005	11:23:27 AM	33.718	33.679	0.039
10:47:39 AM	7.170	7.169	0.001	10:47:39 AM	12.600	12.626	-0.026	L:24:27 AM	28.997	29.007	-0.010	11:24:27 AM	33.699	33.658	0.033
10:48:39 AM	7.231	7.203	0.028	10:48:39 AM	12.518	12.557	-0.039	L:25:27 AM	28.999	29.007	-0.010	11:25:27 AM	33.679	33.642	0.041
10:49:39 AM	7.198	7.196	0.002	10:49:39 AM	12.453	12.484	-0.031	1:26:27 AM	29.003	29.009	-0.010	11:25:27 AM	33.648	33.615	0.037
10:50:39 AM	7.206	7.199	0.007	10:50:39 AM	12.351	12.392	-0.041	L:27:27 AM	29.003	29.009	-0.006	11:27:27 AM	33.628	33.586	0.055
10:51:39 AM		7.128	-0.003	10:51:39 AM	12.273	12.305	-0.032	1.27.27 AM	29.007	29.023		11.27.27 AM	33.611	33.564	0.042

Table 8: Verification of temperature sensors

L:28:27 AM

L:29:27 AM

L:30:27 AM

L:31:27 AM

L:32:27 AM

29.018

29.016

29.023

29.027

29.022

29.040

29.046

29.054

29.065

29.062

33.611

33.568

33.549

33.498

33.496

33.743

11:28:27 AM

11:29:27 AM

11:30:27 AM

11:31:27 AM

11:32:27 AM

33.564

33.535

33.511

33.463

33.462

33.706

7.098

7.031

7.060

7.028

10:52:39 AM

10:53:39 AM

10:54:39 AM

10:55:39 AM

10:56:39 AM

Average

7.085

7.048

7.047

7.032

10:52:39 AM

10:53:39 AM

10:54:39 AM

10:55:39 AM

10:56:39 AM

12.201

12.124

12.106

12.100

12.305 12.239 12.161

12.131

12.129

APPENDIX C

Total Cooling System Efficiency (worked example)

Average reading	Pe	riod	Unit
	Daytime^	Night-time~	
Overall airside efficiency**			kW/RT
Overall waterside efficiency			kW/RT
Total cooling system efficiency			kW/RT

Table 9: Total Cooling System Efficiency (including airside)

[^] For hotels and other developments with 24-hour operations, day-time shall refer to the period from 7am – 11pm; for all other developments, daytime shall refer to the normal operating hours stipulated in clause 6.1.4 of the PEA code

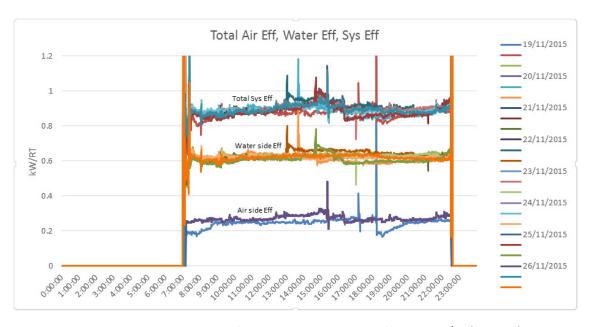


Figure 21: Super-imposed plot of daily total cooling system efficiency kW/RT (Example)

^{**} Applicable for projects certified under NRB 2015 with effect from 1 December 2016

[~]For hotels and other developments with 24-hour operations only; Night-time shall refer to the period from 11pm – 7am;